

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

### **CROSS REFERENCE TO RELATED APPLICATIONS**

This application claims the benefit of U.S. Provisional Application No. 60/549,184, filed March 2, 2004, whose entire contents are hereby incorporated by reference.

### **BACKGROUND OF THE INVENTION**

It is known to provide molded plastic taps for use with containers, in particular disposable containers of the type popular for supplying liquid such as water, wine or milk. One well known type of tap for this purpose is a so-called push button tap having a resilient plastic diaphragm which, when pressed, opens the valve to allow liquid to flow from the container. The resilient plastic diaphragm, commonly referred to as a "push button," can be arranged so that it positively urges the valve into a sealing position when manual pressure is removed therefrom. The tap is therefore self-closing.

An alternative to push button taps are the so-called "rotary" taps. In these, a cap is rotated to in turn rotate a stem within the tap body. Rotation of the stem causes it to uncover an aperture provided in the tap body through which or from which liquid is dispensed.

Irrespective of the type of tap used with a container, it has been found that smooth liquid flow with a stabilized flow profile can only be achieved if either the container is flexible, collapsing as liquid is dispensed, or the container is vented. The reason for this is that otherwise air must flow into the container to fill the space from which liquid has been vacated and equalize the pressure within the container. The inflow of air disrupts the outflow of liquid causing it to be uneven and reducing the flow rate.

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

## SUMMARY OF THE INVENTION

Disclosed herein are air-vented closures for a fluid container, each closure having a dedicated liquid conduit and a dedicated air conduit. This allows air to flow into the container without encountering static or flowing liquid in the air conduit.

In an embodiment, an air-vented closure has a body having a docking member for connecting the closure to a container. The body has a first conduit and a second conduit, the first conduit being adapted for conveying liquid and having a liquid outlet, the second conduit being adapted for conveying air and having an air inlet. The closure also has a member having opposed first and second ends with a liquid outlet at the first end and an air inlet at the second end. The member is positionable with respect to the body from a closed position where no liquid flows through the first conduit to an open position where liquid can flow through the first conduit.

In another embodiment, the closure assembly has a valve body and a valve element. The valve body has a first fluid conduit and a second fluid conduit spaced from the first conduit. The valve body has a mounting sleeve in fluid communication with the first fluid conduit and the second fluid conduit, the mounting sleeve has an axis therethrough. The valve member may be positioned in the mounting sleeve for reciprocating movement therein from a closed position to an open position in response to rotation of the valve member about the axis. The valve member has a wall having a first end and an opposed second end, the valve member having a third fluid conduit therethrough. A first portion of the wall of the valve member may be removed to define an air inlet into the third fluid conduit and a second portion may be removed to define an air outlet from the third conduit. When the valve member is in the closed position a

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

portion of the valve member blocks fluid flow through the first conduit and a portion of the mounting sleeve blocks air flow from the air outlet. When the valve member is in the open position, fluid can flow through the first conduit and air can flow through the air outlet.

Also disclosed herein is a fluid container having an air vented closure attached thereto.

#### **BRIEF DESCRIPTION OF THE FIGURES**

**FIG. 1** is an isometric view of a closure assembly of the present invention;

**FIG. 2** is an end view of the closure of **FIG. 1**;

**FIG. 3** is a side view in partial cross-section of the closure of **FIG. 1**;

**FIG. 4** is a plan view in cross-section of the closure assembly taken along line X-X of **FIG. 3**;

**FIG. 5** is a fluid container with the closure assembly of **FIG. 1**;

**FIG. 6** is a side view in partial cross-section of the closure assembly in a closed position;

**FIG. 7** is a side view in partial cross-section of the closure assembly in an open position;

**FIG. 8** is a schematic view of an embodiment of an air vent of a valve element in an open position;

**FIG. 9** is a schematic view of an embodiment of an air vent of a valve element in an open position;

**FIG. 10** is a schematic view of an embodiment of an air vent of a valve element in an open position;

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

**FIG. 11** is a schematic view of an embodiment of an air vent of a valve element in an open position;

**FIG. 12** is a schematic view of an embodiment of an air vent of a valve element in an open position;

**FIG. 13** is a plot of the area of outlet vs. number of turns of valve element of FIG. 12;

**FIG. 14** is a schematic view of an embodiment of an air vent of a valve element in an open position;

**FIG. 15** is a plot of flow rate vs. time showing a discontinuous flow rate;

**FIG. 16** is a plot of flow rate vs. time for a continuous flow rate;

**FIG. 17** is a cross-sectional view of another embodiment of an air-vented liquid valve in a closed position;

**FIG. 18** is a cross-sectional view of the valve of FIG. 17 in the open position;

**FIG. 19** is a cross-sectional view of another embodiment of an air-vented liquid valve in a closed position;

**FIG. 20** is a cross-sectional view of the valve of FIG. 19 in an open position; and

**FIG. 21** is an end view of a valve element of the valve of FIG. 19.

## **DETAILED DESCRIPTION OF THE INVENTION**

It should be understood that various changes and modifications to the presently preferred embodiments described herein will be apparent to those skilled in the art. Such changes and modifications can be made without departing from the spirit and scope of the

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

present invention and without diminishing its intended advantages. It is therefore intended that such changes and modifications be covered by the appended claims.

Referring now to FIGS. 1 to 4, a closure assembly 10 having a valve body 12 and a valve member 14 is shown. The valve body 12 has a docking member 16 an annular flange 18 and a mounting sleeve 20. The docking member 16 is for connecting the assembly 10 to a container 22 (FIG. 5). The annular flange 18 defines a first fluid conduit 24 and a second air conduit 26 extending parallel to one another. The mounting sleeve 20 defines a fluid channel 28 having an axis 30. The fluid channel 28 is dimensioned to coaxially receive the valve member 14. As will be described in greater detail herein, the valve member 14 is moveable from a closed position to an open position to allow liquid to flow outward from the container through the first fluid conduit 24 while air flows into the container through the second air conduit 26 without having to pass through a static or flowing liquid in the conduit.

The valve body 12 is preferably made from a polymeric material and is manufactured by a polymer processing technique. In a preferred form, the valve body is manufactured by injection molding. The first fluid conduit 24 and the second air conduit 26 are separated by a wall 32. The wall 32 divides an internal pathway of the annular flange 18 into conduits. The first liquid conduit 24 and the second air conduit 26 are shown having differing volumes yet the invention contemplates having the first conduit and second conduit having the same or approximately the same volume. In a preferred form of the invention, the volume of the first fluid conduit 24 has a ratio with respect to the second air conduit 26 of from about 0.3 - 4.0 and more preferably from 0.5 - 3.0. The

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

first conduit 24 has a fluid inlet end 40 and a fluid outlet 42. The second conduit 26 has a first air inlet 44 and a first air outlet 46.

The mounting sleeve 20 has a generally cylindrically shaped wall having a first end 50, a second end 52 and an outer surface 54. A pair of circumferentially spaced, spiral extending grooves 56 extend from an intermediate portion of the mounting sleeve to proximate the first end 50. The grooves 56 are shown extending through the entire thickness of the sleeve 20. However, it is contemplated that grooves 56 can be provided on an interior surface of the sleeve 20 that do not extend through the entire thickness of the sidewall (less than 98% of the thickness) so that the grooves are hidden from view. The groove has a top edge 58 and a bottom edge 60 and top stop 62 and a bottom stop 64. A protuberance 66 extends from the top edge 60 proximate the bottom stop 64. A gap 68 separates the protuberance 66 from the bottom stop 64. The second end 52 of the sleeve 20 has a spout 69 having a taper 70 defining a reduced diameter portion when compared to the diameter of the remainder of the sleeve 20.

The valve element 14 has a first end 80 and a second end 82. The valve element 14 has a generally cylindrically shaped side wall having an outer surface, a gripping projection 86 at the first end 80 and a pair of circumferentially spaced pins 88. The pins 88 fit within the grooves 56 of the valve body. Rotation of the valve element 14 about the axis 30 causes reciprocating movement of the valve element 14 along the axis 30. A second air outlet 94 is formed in the side wall proximate the second end 82. When in the open position the second air outlet 94 is in alignment with the air conduit 26, but not in alignment when in the closed position. FIG. 6 shows the valve element 14 in the closed position and FIG. 7 shows valve element 14 in an open position. The protuberance 66

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

holds the valve element in the closed position to prevent inadvertent dispensing of liquid from the container. A force that can be generated by hand is sufficient to overcome the resistance of the protuberance to rotation of the valve element 14.

In a preferred form of the invention the ratio of volumes of the second air outlet ~~(not shown)~~ 94 and the opening of the air conduit 26 and the configuration of the second air outlet 94 and the first air inlet 44 are selected to minimize the vacuum drawn on the container contents when activating flow of fluid through the spout. It is also desirable to provide a continuous flow during dispensing to 30 minimize or eliminate interrupted flow from the container causing a familiar "glug" sound. In another preferred form of the invention, a water-filled 1 1/2 gallon rigid container can be continuously dispensed (See FIGS. 17, 18) without interruption until the container is drained.

The valve element 14 has a second air inlet 92 on an end opposite the second air outlet 94, a third conduit 95 defined as running through the valve element 14 from the second air inlet 92 to the second air outlet 94. It is contemplated positioning the inlet 92 on the side wall proximate the first end 50 so that the inlet 92 is covered by the mounting sleeve when the element is in the closed position and is uncovered when moved into the open position. The inlet 92 is open to ambient air. It is contemplated closing the inlet 92 with a valve, such as a flapper valve, which would open when the valve element is in the open position.

FIG. 5 shows the assembly 10 mounted to a container 22. The container can be made from polymeric materials, paperboard, or metal. In a preferred form, the container is a polymeric material shaped into a container by any suitable polymer processing techniques such as injection molding, blow molding, by sealing sheets of material

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

together to define a container or other suitable process. Suitable polymers include, but are not limited to, homopolymers and copolymers of polyolefins, polyamides, polyesters or other suitable material. One particularly suitable material is a homopolymer of ethylene and more preferably one having a density of greater than about 0.915 g/cc. In another embodiment, the material is an HDPE. In a preferred container, the sidewalls will have a modulus of elasticity of greater than 20,000 psi. In another preferred form of the container, the sidewalls of the container will not substantially collapse upon draining the contents of the container.

The configuration of the second air outlet 94 and air inlet 44 can take on many forms as shown in representative embodiments shown in FIGS. 8 - 12 and 14. The shape and size of the air inlet 44 and/or second air outlet 94 can take on numerous forms including circular, semi-circular, oval, polygonal, irregular or amorphous. The second air outlet 94 can also be divided into separate chambers by a dividing wall extending between or within the internal surfaces of the first air inlet 44. It is also contemplated the air inlet 44 may terminate with a wall having a singular outlet having one of the many shapes set forth above or have a series of sub-outlets of any shape or combination of shapes.

For the sake of brevity, FIGS. 8 - 12, and 14 are shown with an air inlet 44 that is semicircular in shape with the valve element 14 in the full open position. It should be understood the semi-circular shape of the air inlet 44 can be replaced by any one of the shapes or configurations described above. FIG. 8 shows the second air outlet 94 having three circular shaped sub-outlets 100 each of approximately equal area form a triangular shape, and particularly an equilateral triangle. Thus, the sub-outlets can be positioned to



Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

form a pattern that is circular, semi-circular, oval, polygonal, irregular or amorphous.

When moving the valve element from a closed position to an open position, the first sub-outlet **104** comes into alignment with the air inlet **44** followed by the second sub-outlet **106** and then the third sub-outlet **108**. The first sub-outlet comes into alignment with the air inlet **44** at it is positioned higher on the valve element than the other sub-outlets. The second sub-outlet is on a leading edge **110** of the valve element, and, therefore a leading edge portion **107** of the sub-outlet **106** initially comes into alignment with the air inlet **44** and then is joined by the third sub-outlet positioned on a trailing edge **112** of the valve element.

FIG. 9 shows a similar configuration of sub-outlets as FIG. 8 with the exception that the first sub-outlet **104** has a greater area than the second and third sub-outlets **106**, **108**.

FIGS. 10 and 11 show a valve element having two-sub-outlets **104** and **106**. FIG. 10 shows the first sub-outlet **104** positioned above the second sub-outlet. The distance between the first sub-inlet and the second sub-inlet can be traversed upon rotation of the valve element by a number of turns about the axis of from about  $1/8^{\text{th}}$  of a turn to 1 full turn. FIG. 11 shows the first sub-outlet **104** positioned a distance ahead of the second sub-outlet **106** and this distance should be traversed by a number of turns about the axis of from about  $1/8^{\text{th}}$  of a turn to 1 full turn.

FIG. 12 shows a valve element having three sub-outlets having the second sub-outlet **106** spaced a distance  $h1$  and  $w1$  from the first outlet and the third sub-outlet **108** spaced a distance  $h2$  and  $w2$  from the first sub-outlet **104**. Thus the three sub-outlets form a line having a slope  $h2/w2$ .

Replacement Specification – Clean Copy  
App. Ser. No. 10/591,493  
October 24, 2008

FIG. 13 shows a graph of the area of alignment between the air outlet 94 versus the number of turns of the valve element 14 for the embodiment shown in FIG. 12. Initially the valve element is rotated for a lead in section where there is no alignment between the outlet 94 and the inlet 44. As the first sub-outlet 104 comes into alignment with the inlet 44 there is an initial increase in the alignment volume at an increasing rate up to the point where half the first circular sub-outlet 104 has been reached 132, to form a first inflection point, and continues to increase at a declining rate 134 until the first sub-outlet 104 is in alignment with the inlet 44. The inflection point 134 is reached when the valve element has traveled a distance corresponding to  $h1$  in FIG. 12. The area does not change 136 and the curve flattens until the second circular sub-outlet begins to come into alignment with the air inlet 44 and increases similarly 138 as for the first sub-outlet. The third sub-outlet then comes into alignment and also increases the area in a similar fashion 140 as the first and second sub-outlets. By having the sub-outlets positioned on the valve element in these orientations allows for a sequential and discontinuous (interrupted by periods where rotation of the valve element does not increase the area of alignment)

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